

IN THE SPECIFICATION:

Please insert the following on Page 1, line between the title and line 3:

--BACKGROUND OF THE INVENTION

1. Field of the Invention--

Please revise the paragraph beginning on Page 1, line 8 to read as follows:

--To generate a cyclical intake and compression stroke in the individual cylinders of a cylinder drum in an axial piston machine, a swash plate is used, which, on rotation of the cylinder drum, causes the pistons in the individual cylinders to effect a cyclical reciprocating motion by means of guide shoes which are connected to the pistons and are supported on the swash plate, continuously on a circular band concentric to the axis of rotation in accordance with the rotational movement. To ensure defined support of the guide shoes on the swash plate, a pressure acting in the direction of the swash plate is exerted on the return plate carrying the individual guide shoes via a return member, which is connected with the drive shaft of the cylinder drum. To this end, the pretensioning force of a ~~[[tension]]~~ compression spring, which is guided over the drive shaft of the cylinder drum, is applied to the return member via a plurality of pressure pins.--

Please revise the paragraph beginning on Page 1, line 26 to Page 2, line 8 to read as follows:

--Transmission of the pretensioning force from the ~~[[tension]]~~ compression spring via a plurality of pressure pins to the return member has caused problems in the past. Solutions in which the pressure pins are guided in separate grooves in the drive shaft require additional spring retainers for local stabilisation of the pressure pins in the grooves, which complicates the assembly process and also increases unnecessarily the manufacturing costs of the axial piston machine due to the production and storage of additional components. Advanced solutions, in which the pressure pins are fixed for better guidance in grooves with limited lateral dimensions without using additional components, have the disadvantage that the pressure pins are freely rotatable against the surfaces opposite their end faces. To minimise the resultant increased wear, harder materials are required, which likewise increase unnecessarily the manufacturing costs for the axial piston machines.--

Please insert the following on Page 2, between lines 8 and 10:

--2. Discussion of the Prior Art--

Please insert the following on Page 2, between lines 21 and 23:

--SUMMARY OF THE INVENTION--

Please revise the paragraph beginning on Page 2, line 23 to read as follows:

--The object of the invention, therefore, is so to develop ~~[[the]]~~ an axial piston machine having the ~~features according to the precharacterising clause of claim 1 and the features and at least one pressure pin having [[the]] features of the precharacterising clause of claim 15 that such~~ that any wear to the pressure pins and return member as a result of oscillating and micro-movements of the pressure pin against the return member no longer occurs. Moreover, the invention should additionally ~~[[fulfil]]~~ fulfill the requirements resulting from the above-stated problems:

- no use of additional components
- no lateral or radial displacement of the pressure pins after installation
- no rotational movement between end faces of the pressure pins and adjacent surfaces of opposing components (return member, spring washer)
- easy assembly of the components
- economic viability of manufacture
- long service life
- simple construction.--

Please delete the paragraph on Page 3, line 11 through line 14 in its entirety.

Please insert the following on Page 5, between lines 3 and 5:

--BRIEF DESCRIPTION OF THE DRAWINGS--

Please revise the paragraph beginning on Page 7, line 26 to Page 8, line 4 to read as follows:

--The force of the return means 15 acting in the direction of the longitudinal axis 13 of the drive shaft 4 for ensuring that the guide shoes 8 rest securely against the inclined surface 12 is supplied to the return means 15 as the pretensioning force of a pretensioned ~~[[tension]]~~ compression spring 27 via a plurality of pressure pins 28. The ~~[[tension]]~~ compression spring 26 is guided over the drive shaft 4 in a recess in the cylinder drum 5 and is held under tension between a spring ring 29 fixed in the cylinder drum 5 in the region of the control plate 17 and a spring washer 30 guided movably over the drive shaft 4 in the direction of the longitudinal axis 13 thereof in the region of the return means 15.--

Please revise the paragraph beginning on Page 8, line 6 to read as follows:

--To transmit the pretensioning force of the ~~[[tension]]~~ compression spring 27 via the spring washer 30 to the pressure pins 28, each pressure pin 28 comprises according to the invention a surface enlargement 32 at its top end 31 facing the spring washer 30. In the exemplary embodiment, the surface enlargement 32 is a flange pointing radially to one side of the longitudinal axis 33 of the pressure pin 28, which flange expands the end face of the cylindrical basic member 34 by the end face of the surface enlargement 32 likewise of planar construction and pointing in the same direction as the end face of the cylindrical basic member 34, to produce the bearing surface 35. At the outer end of the surface enlargement 32, a pointed retaining hook 36 projects perpendicularly out of the bearing surface 35.--

Please revise the paragraph beginning on Page 9, line 31 to Page 10, line 9 to read as follows:

--Through provision of the enlarged, bent-out bearing surface 45 at the bottom end 40 of the pressure pin 28, which surface 45 preferably corresponds to at least double the original end face of the cylindrical basic member 34, the pretensioning force produced by the ~~[[tension]~~ compression spring 27 and acting via the spring washer 30 on the pressure pins 28 is distributed over a larger surface area, such that the surface pressure exerted by the bearing surface 45 of the pressure pin 28 on the surface 46 of the return member 26 is reduced. The wear suffered by the two surfaces 45 of the pressure pin 28 and 46 of the return member 26 are minimised accordingly during regular operation.--